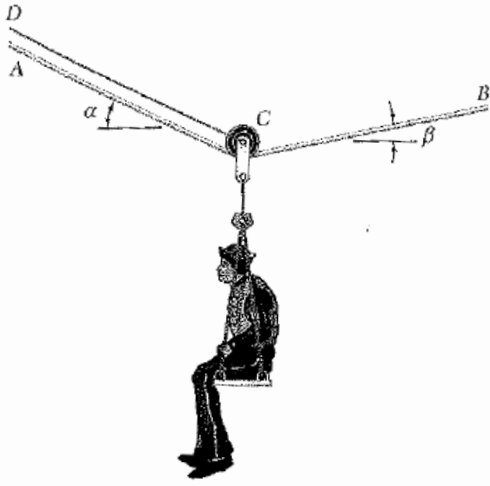


Homework #3 Solution

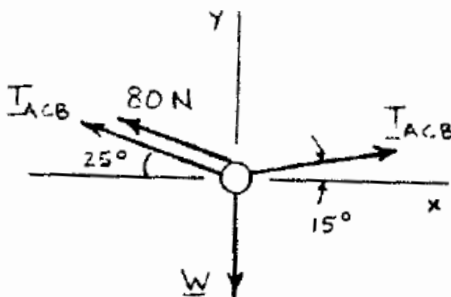
PROBLEM 2.54

A sailor is being rescued using a boatswain's chair that is suspended from a pulley that can roll freely on the support cable ACB and is pulled at a constant speed by cable CD . Knowing that $\alpha = 25^\circ$ and $\beta = 15^\circ$ and that the tension in cable CD is 80 N, determine (a) the combined weight of the boatswain's chair and the sailor, (b) in tension in the support cable ACB .



SOLUTION

Free-Body Diagram



$$\pm \Sigma F_x = 0: T_{ACB} \cos 15^\circ - T_{ACB} \cos 25^\circ - (80 \text{ N}) \cos 25^\circ = 0$$

$$T_{ACB} = 1216.15 \text{ N}$$

$$+\uparrow \Sigma F_y = 0: (1216.15 \text{ N}) \sin 15^\circ + (1216.15 \text{ N}) \sin 25^\circ$$

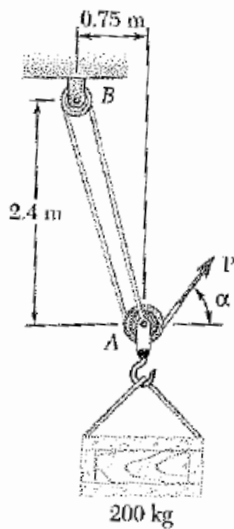
$$+ (80 \text{ N}) \sin 25^\circ - W = 0$$

$$W = 862.54 \text{ N}$$

(a) $W = 863 \text{ N}$ ◀

(b) $T_{ACB} = 1216 \text{ N}$ ◀

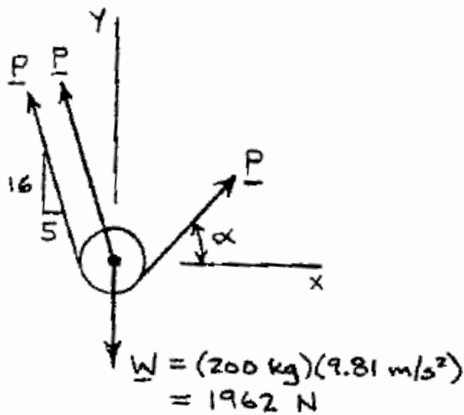
PROBLEM 2.66



A 200-kg crate is to be supported by the rope-and-pulley arrangement shown. Determine the magnitude and direction of the force \mathbf{P} that must be exerted on the free end of the rope to maintain equilibrium. (*Hint:* The tension in the rope is the same on each side of a simple pulley. This can be proved by the methods of Ch. 4.)

SOLUTION

Free-Body Diagram: Pulley A



$$+\rightarrow \Sigma F_x = 0: \quad -2P \left(\frac{5}{\sqrt{281}} \right) + P \cos \alpha = 0$$

$$\cos \alpha = 0.59655$$

$$\alpha = \pm 53.377^\circ$$

For $\alpha = +53.377^\circ$:

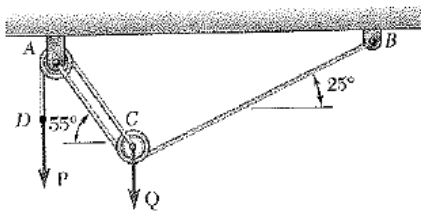
$$+\uparrow \Sigma F_y = 0: \quad 2P \left(\frac{16}{\sqrt{281}} \right) + P \sin 53.377^\circ - 1962 \text{ N} = 0$$

$$\mathbf{P} = 724 \text{ N} \swarrow 53.4^\circ \blacktriangleleft$$

For $\alpha = -53.377^\circ$:

$$+\uparrow \Sigma F_y = 0: \quad 2P \left(\frac{16}{\sqrt{281}} \right) + P \sin(-53.377^\circ) - 1962 \text{ N} = 0$$

$$\mathbf{P} = 1773 \text{ N} \searrow 53.4^\circ \blacktriangleleft$$

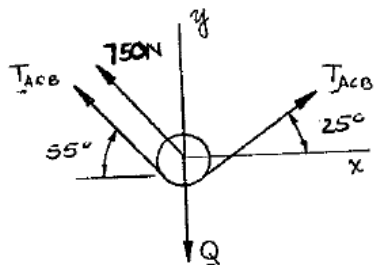


PROBLEM 2.69

A load Q is applied to the pulley C , which can roll on the cable ACB . The pulley is held in the position shown by a second cable CAD , which passes over the pulley A and supports a load P . Knowing that $P = 750$ N, determine (a) the tension in cable ACB , (b) the magnitude of load Q .

SOLUTION

Free-Body Diagram: Pulley C



$$(a) \quad \pm \Sigma F_x = 0: \quad T_{ACB}(\cos 25^\circ - \cos 55^\circ) - (750 \text{ N})\cos 55^\circ = 0$$

$$\text{Hence:} \quad T_{ACB} = 1292.88 \text{ N}$$

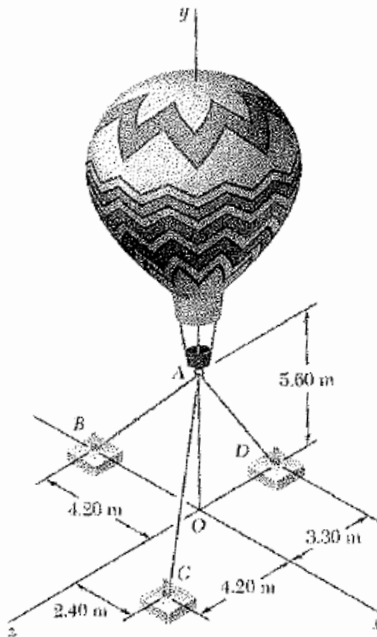
$$T_{ACB} = 1293 \text{ N} \quad \blacktriangleleft$$

$$(b) \quad +\uparrow \Sigma F_y = 0: \quad T_{ACB}(\sin 25^\circ + \sin 55^\circ) + (750 \text{ N})\sin 55^\circ - Q = 0$$

$$(1292.88 \text{ N})(\sin 25^\circ + \sin 55^\circ) + (750 \text{ N})\sin 55^\circ - Q = 0$$

$$\text{or} \quad Q = 2219.8 \text{ N} \quad Q = 2220 \text{ N} \quad \blacktriangleleft$$

PROBLEM 2.102



Three cables are used to tether a balloon as shown. Knowing that the balloon exerts an 800-N vertical force at A, determine the tension in each cable.

SOLUTION

See Problem 2.101 for the figure and analysis leading to the linear algebraic Equations (1), (2), and (3).

$$-0.6T_{AB} + 0.32432T_{AC} = 0 \quad (1)$$

$$-0.8T_{AB} - 0.75676T_{AC} - 0.86154T_{AD} + P = 0 \quad (2)$$

$$0.56757T_{AC} - 0.50769T_{AD} = 0 \quad (3)$$

From Eq. (1): $T_{AB} = 0.54053T_{AC}$

From Eq. (3): $T_{AD} = 1.11795T_{AC}$

Substituting for T_{AB} and T_{AD} in terms of T_{AC} into Eq. (2) gives

$$-0.8(0.54053T_{AC}) - 0.75676T_{AC} - 0.86154(1.11795T_{AC}) + P = 0$$

$$2.1523T_{AC} = P; \quad P = 800 \text{ N}$$

$$T_{AC} = \frac{800 \text{ N}}{2.1523}$$

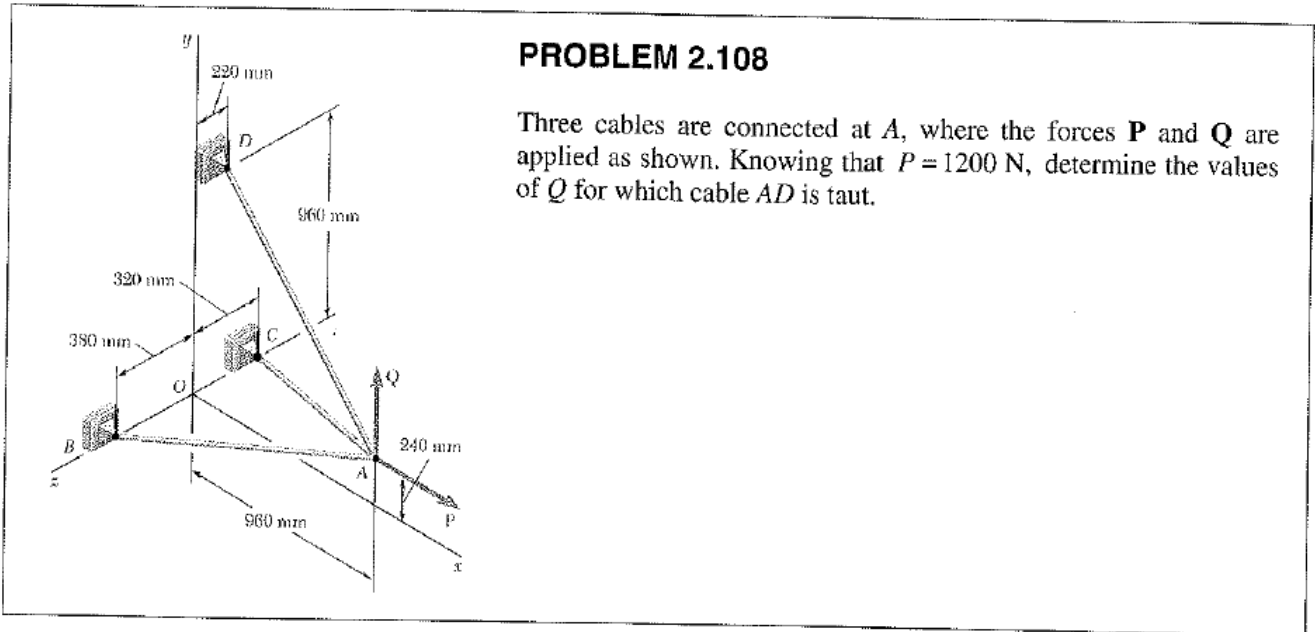
$$= 371.69 \text{ N}$$

Substituting into expressions for T_{AB} and T_{AD} gives

$$T_{AB} = 0.54053(371.69 \text{ N})$$

$$T_{AD} = 1.11795(371.69 \text{ N})$$

$$T_{AB} = 201 \text{ N}, \quad T_{AC} = 372 \text{ N}, \quad T_{AD} = 416 \text{ N} \quad \blacktriangleleft$$



PROBLEM 2.108

Three cables are connected at A, where the forces **P** and **Q** are applied as shown. Knowing that $P = 1200$ N, determine the values of Q for which cable AD is taut.

SOLUTION

We assume that $T_{AD} = 0$ and write $\Sigma \mathbf{F}_A = 0: \mathbf{T}_{AB} + \mathbf{T}_{AC} + Q\mathbf{j} + (1200 \text{ N})\mathbf{i} = 0$

$$\overline{AB} = -(960 \text{ mm})\mathbf{i} - (240 \text{ mm})\mathbf{j} + (380 \text{ mm})\mathbf{k} \quad AB = 1060 \text{ mm}$$

$$\overline{AC} = -(960 \text{ mm})\mathbf{i} - (240 \text{ mm})\mathbf{j} - (320 \text{ mm})\mathbf{k} \quad AC = 1040 \text{ mm}$$

$$\mathbf{T}_{AB} = T_{AB} \lambda_{AB} = T_{AB} \frac{\overline{AB}}{AB} = \left(-\frac{48}{53}\mathbf{i} - \frac{12}{53}\mathbf{j} + \frac{19}{53}\mathbf{k} \right) T_{AB}$$

$$\mathbf{T}_{AC} = T_{AC} \lambda_{AC} = T_{AC} \frac{\overline{AC}}{AC} = \left(-\frac{12}{13}\mathbf{i} - \frac{3}{13}\mathbf{j} - \frac{4}{13}\mathbf{k} \right) T_{AC}$$

Substituting into $\Sigma \mathbf{F}_A = 0$, factoring **i**, **j**, **k**, and setting each coefficient equal to ϕ gives:

$$\mathbf{i}: -\frac{48}{53}T_{AB} - \frac{12}{13}T_{AC} + 1200 \text{ N} = 0 \quad (1)$$

$$\mathbf{j}: -\frac{12}{53}T_{AB} - \frac{3}{13}T_{AC} + Q = 0 \quad (2)$$

$$\mathbf{k}: \frac{19}{53}T_{AB} - \frac{4}{13}T_{AC} = 0 \quad (3)$$

Solving the resulting system of linear equations using conventional algorithms gives:

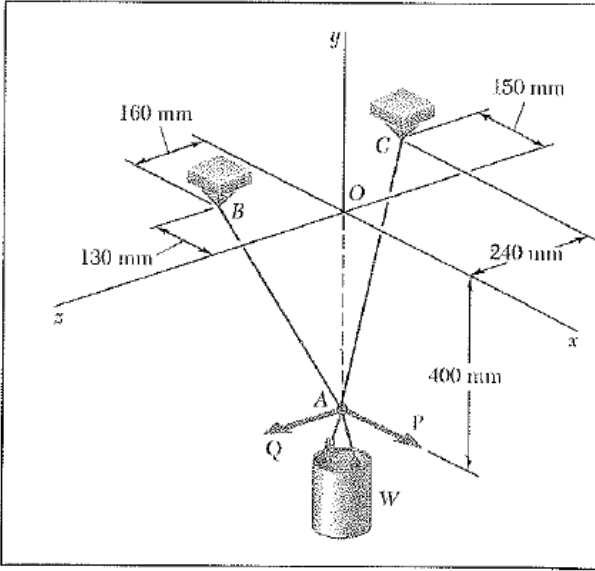
$$T_{AB} = 605.71 \text{ N}$$

$$T_{AC} = 705.71 \text{ N}$$

$$Q = 300.00 \text{ N}$$

$$0 \leq Q < 300 \text{ N} \blacktriangleleft$$

Note: This solution assumes that Q is directed upward as shown ($Q \geq 0$), if negative values of Q are considered, cable AD remains taut, but AC becomes slack for $Q = -460$ N.



PROBLEM 2.123

A container of weight W is suspended from ring A . Cable BAC passes through the ring and is attached to fixed supports at B and C . Two forces $\mathbf{P} = P\mathbf{i}$ and $\mathbf{Q} = Q\mathbf{k}$ are applied to the ring to maintain the container in the position shown. Knowing that $W = 376 \text{ N}$, determine P and Q . (Hint: The tension is the same in both portions of cable BAC .)

SOLUTION

$$\begin{aligned} \mathbf{T}_{AB} &= T\lambda_{AB} \\ &= T \frac{\overline{AB}}{AB} \\ &= T \frac{(-130 \text{ mm})\mathbf{i} + (400 \text{ mm})\mathbf{j} + (160 \text{ mm})\mathbf{k}}{450 \text{ mm}} \\ &= T \left(-\frac{13}{45}\mathbf{i} + \frac{40}{45}\mathbf{j} + \frac{16}{45}\mathbf{k} \right) \end{aligned}$$

$$\begin{aligned} \mathbf{T}_{AC} &= T\lambda_{AC} \\ &= T \frac{\overline{AC}}{AC} \\ &= T \frac{(-150 \text{ mm})\mathbf{i} + (400 \text{ mm})\mathbf{j} + (-240 \text{ mm})\mathbf{k}}{490 \text{ mm}} \\ &= T \left(-\frac{15}{49}\mathbf{i} + \frac{40}{49}\mathbf{j} - \frac{24}{49}\mathbf{k} \right) \end{aligned}$$

$$\Sigma \mathbf{F} = 0: \quad \mathbf{T}_{AB} + \mathbf{T}_{AC} + \mathbf{Q} + \mathbf{P} + \mathbf{W} = 0$$

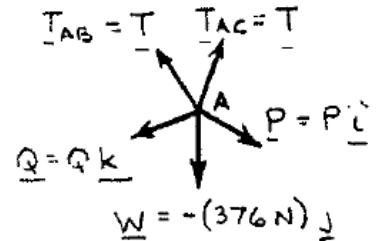
Setting coefficients of \mathbf{i} , \mathbf{j} , \mathbf{k} equal to zero:

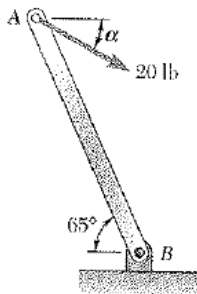
$$\mathbf{i}: \quad -\frac{13}{45}T - \frac{15}{49}T + P = 0 \qquad 0.59501T = P \qquad (1)$$

$$\mathbf{j}: \quad +\frac{40}{45}T + \frac{40}{49}T - W = 0 \qquad 1.70521T = W \qquad (2)$$

$$\mathbf{k}: \quad +\frac{16}{45}T - \frac{24}{49}T + Q = 0 \qquad 0.134240T = Q \qquad (3)$$

Free-Body A:



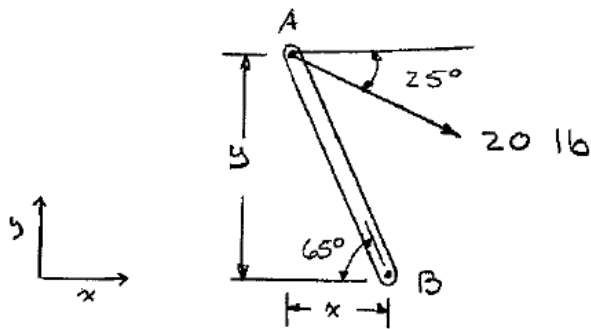


PROBLEM 3.1

A 20-lb force is applied to the control rod AB as shown. Knowing that the length of the rod is 9 in. and that $\alpha = 25^\circ$, determine the moment of the force about Point B by resolving the force into horizontal and vertical components.

SOLUTION

Free-Body Diagram of Rod AB :



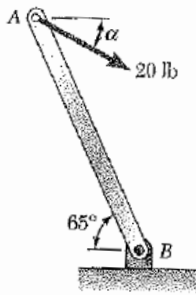
$$\begin{aligned} x &= (9 \text{ in.}) \cos 65^\circ \\ &= 3.8036 \text{ in.} \\ y &= (9 \text{ in.}) \sin 65^\circ \\ &= 8.1568 \text{ in.} \end{aligned}$$

$$\begin{aligned} \mathbf{F} &= F_x \mathbf{i} + F_y \mathbf{j} \\ &= (20 \text{ lb} \cos 25^\circ) \mathbf{i} + (-20 \text{ lb} \sin 25^\circ) \mathbf{j} \\ &= (18.1262 \text{ lb}) \mathbf{i} - (8.4524 \text{ lb}) \mathbf{j} \end{aligned}$$

$$\mathbf{r}_{A/B} = \overrightarrow{BA} = (-3.8036 \text{ in.}) \mathbf{i} + (8.1568 \text{ in.}) \mathbf{j}$$

$$\begin{aligned} \mathbf{M}_B &= \mathbf{r}_{A/B} \times \mathbf{F} \\ &= (-3.8036 \mathbf{i} + 8.1568 \mathbf{j}) \times (18.1262 \mathbf{i} - 8.4524 \mathbf{j}) \\ &= 32.150 \mathbf{k} - 147.852 \mathbf{k} \\ &= -115.702 \text{ lb-in.} \end{aligned}$$

$$\mathbf{M}_B = 115.7 \text{ lb-in.} \quad \blacktriangleleft$$

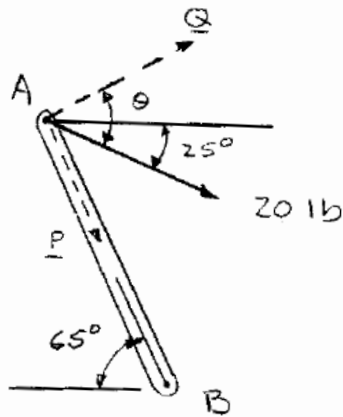


PROBLEM 3.2

A 20-lb force is applied to the control rod AB as shown. Knowing that the length of the rod is 9 in. and that $\alpha = 25^\circ$, determine the moment of the force about Point B by resolving the force into components along AB and in a direction perpendicular to AB .

SOLUTION

Free-Body Diagram of Rod AB :

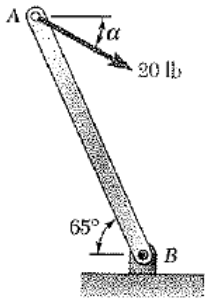


$$\begin{aligned}\theta &= 90^\circ - (65^\circ - 25^\circ) \\ &= 50^\circ\end{aligned}$$

$$\begin{aligned}Q &= (20 \text{ lb}) \cos 50^\circ \\ &= 12.8558 \text{ lb}\end{aligned}$$

$$\begin{aligned}M_B &= Q(9 \text{ in.}) \\ &= (12.8558 \text{ lb})(9 \text{ in.}) \\ &= 115.702 \text{ lb-in.}\end{aligned}$$

$$M_B = 115.7 \text{ lb-in.} \quad \blacktriangleleft$$

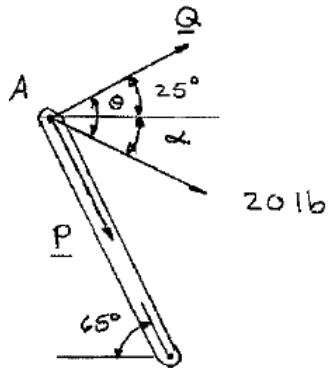


PROBLEM 3.3

A 20-lb force is applied to the control rod AB as shown. Knowing that the length of the rod is 9 in. and that the moment of the force about B is 120 lb·in. clockwise, determine the value of α .

SOLUTION

Free-Body Diagram of Rod AB :



$$\alpha = \theta - 25^\circ$$

$$Q = (20 \text{ lb}) \cos \theta$$

and $M_B = (Q)(9 \text{ in.})$

Therefore, $120 \text{ lb-in.} = (20 \text{ lb})(\cos \theta)(9 \text{ in.})$

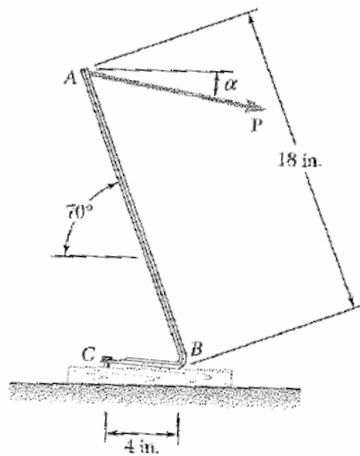
$$\cos \theta = \frac{120 \text{ lb-in.}}{180 \text{ lb-in.}}$$

or $\theta = 48.190^\circ$

Therefore, $\alpha = 48.190^\circ - 25^\circ$

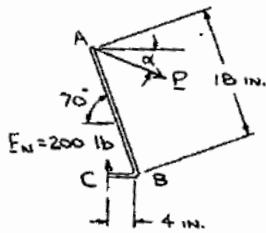
$$\alpha = 23.2^\circ \blacktriangleleft$$

PROBLEM 3.8



It is known that a vertical force of 200 lb is required to remove the nail at C from the board. As the nail first starts moving, determine (a) the moment about B of the force exerted on the nail, (b) the magnitude of the force P that creates the same moment about B if $\alpha = 10^\circ$, (c) the smallest force P that creates the same moment about B.

SOLUTION



(a) We have $M_B = r_{CB} F_N$
 $= (4 \text{ in.})(200 \text{ lb})$
 $= 800 \text{ lb} \cdot \text{in.}$

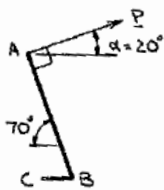
or $M_B = 800 \text{ lb} \cdot \text{in.}$ ◀



(b) By definition, $M_B = r_{AB} P \sin \theta$
 $\theta = 10^\circ + (180^\circ - 70^\circ)$
 $= 120^\circ$

Then $800 \text{ lb} \cdot \text{in.} = (18 \text{ in.}) \times P \sin 120^\circ$

or $P = 51.3 \text{ lb}$ ◀



(c) For P to be minimum, it must be perpendicular to the line joining Points A and B. Thus, P must be directed as shown.

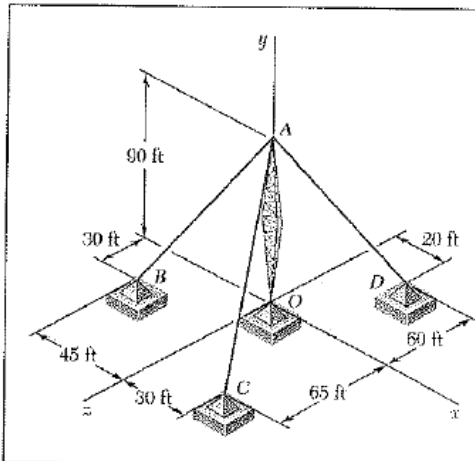
Thus $M_B = d P_{\min}$
 $d = r_{AB}$

or $800 \text{ lb} \cdot \text{in.} = (18 \text{ in.}) P_{\min}$

or $P_{\min} = 44.4 \text{ lb}$

$P_{\min} = 44.4 \text{ lb}$ ◀ 20°

Bonus



PROBLEM 2.112

A transmission tower is held by three guy wires attached to a pin at A and anchored by bolts at B , C , and D . If the tension in wire AC is 920 lb, determine the vertical force P exerted by the tower on the pin at A .

SOLUTION

See Problem 2.111 for the figure and the analysis leading to the linear algebraic Equations (1), (2), and (3) below:

$$-\frac{3}{7}T_{AB} + \frac{6}{23}T_{AC} + \frac{2}{11}T_{AD} = 0 \quad (1)$$

$$-\frac{6}{7}T_{AB} - \frac{18}{23}T_{AC} - \frac{9}{11}T_{AD} + P = 0 \quad (2)$$

$$\frac{2}{7}T_{AB} + \frac{13}{23}T_{AC} - \frac{6}{11}T_{AD} = 0 \quad (3)$$

Substituting for $T_{AC} = 920$ lb in Equations (1), (2), and (3) above and solving the resulting set of equations using conventional algorithms gives:

$$-\frac{3}{7}T_{AB} + 240 \text{ lb} + \frac{2}{11}T_{AD} = 0 \quad (1')$$

$$-\frac{6}{7}T_{AB} - 720 \text{ lb} - \frac{9}{11}T_{AD} + P = 0 \quad (2')$$

$$\frac{2}{7}T_{AB} + 520 \text{ lb} - \frac{6}{11}T_{AD} = 0 \quad (3')$$

Solving,

$$T_{AB} = 1240.00 \text{ lb}$$

$$T_{AD} = 1602.86 \text{ lb}$$

$$P = 3094.3 \text{ lb}$$

$$P = 3090 \text{ lb} \quad \blacktriangleleft$$